

A photograph of an industrial facility, likely a power plant or manufacturing plant. The upper portion of the image shows a large metal structure with six large, circular fans mounted on a grid. Below this, the facility is filled with complex machinery, including large green industrial units, pipes, and electrical control panels. The scene is brightly lit, possibly by natural light from an open structure.

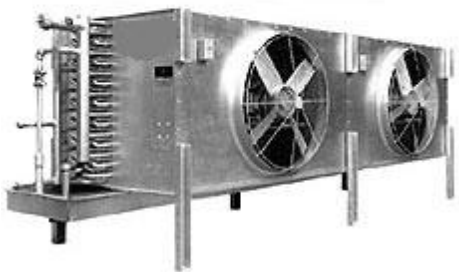
Midwest Mechanical Services & Solutions

2324 Centerline Industrial Drive - St. Louis, MO 63146 - 314-707-7655 - www.midweststl.com



Refrigeration

Understanding Relationships

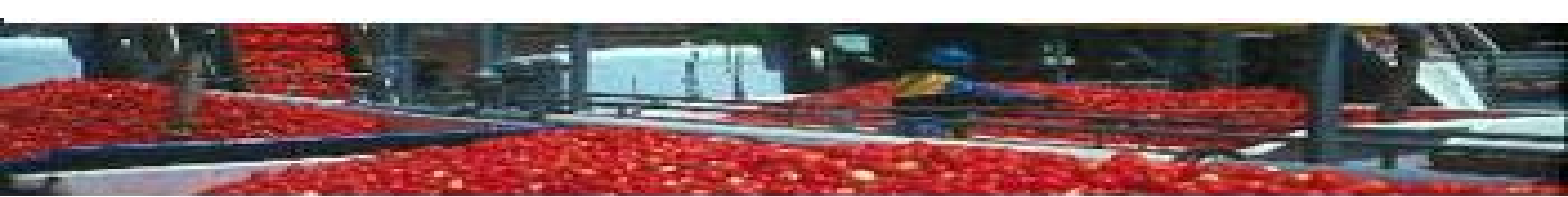




Refrigeration Efficiency =



**Precise Load Matching +
Lower Compression Ratios**



Basic Approach Strategies:

- **Capacity Requirements based on intended use in lieu of “Rule of Thumb” sizing.**
- **Compressor Selection Based on Year-Round Load Characteristics.**
- **Condenser Capacity Optimization**
- **Low Pressure Drop Piping Design**
- **Third Power Fan and Pump Laws**
- **Floating but Stable Discharge Pressures**
- **Floating Suction Pressures**
- **Premium High Efficiency Motors**
- **Integrated and Flexible PLC Control Systems.**

MIDWEST MECHANICAL WANTS TO EARN YOUR REFRIGERATION BUSINESS:

- SERVICE
- PARTS
- UPGRADES
- NEW PROJECTS

AMMONIA & FREON
REFRIGERATION SYSTEMS

How We Can Save You Money:

- Improved Production
- Increased "Up Time"
- Reduction in Energy Costs
- Demand Cost Avoidance
- Lower Maintenance Costs
- FREE Mechanical Risk Index (MRI)
- Records of Performance
- Increased Profits and Customer Satisfaction
- Improved Utilization of Resources



CONTACTS:

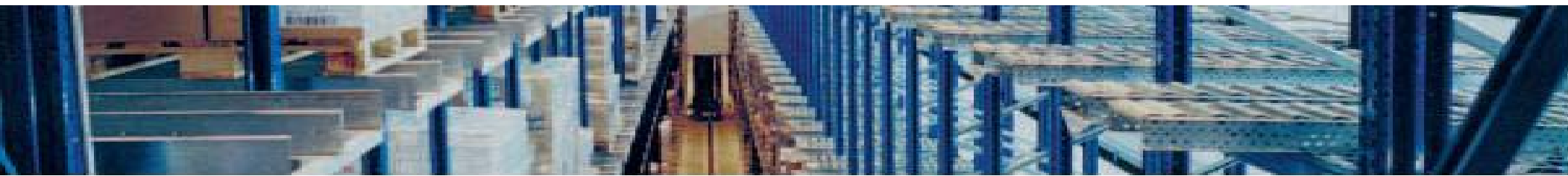
MIKE CASSANO - INDUSTRIAL REFRIGERATION MANAGER
E-Mail: MCassano@midweststl.com

MATT RICHARDSON - VICE PRESIDENT OF OPERATIONS
E-Mail: MRichardson@midweststl.com





Refrigeration industry characteristics



- **Large energy users, high load factor**
- **Facilities expanded over time, always evolving**
- **Risk averse, slow to change**
 - ◆ high cost of investments and mistakes
 - ◆ experience takes time
 - ◆ heavy reliance on plant operators, transferred knowledge
- **No national/state ratings or standards for refrigeration equipment**
- **Systems not “packaged”**
 - ◆ systems are built of components
 - ◆ every application and each system is different
- **Minimal A&E influence**

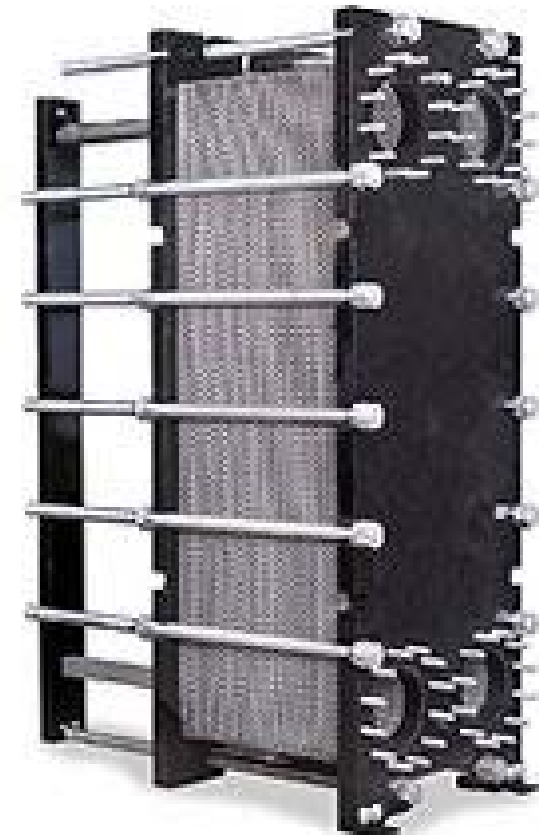


Industrial refrigeration systems themes that impact energy efficiency





- **Large difference in design vs. average load**
- **Systems operated at fixed settings**
- **Characteristically difficult to control**
 - ◆ Relatively unstable due to low circuit mass
- **Slide valve losses are a large inefficiency**
- **Multiple compressor sequencing challenges**
 - ◆ Typical over-control (hunting) vs. actual load needs
 - ◆ Manual sequence control is common
 - ◆ No well developed methodology or control theory
- **25%+ savings potential (actual vs. ideal)**

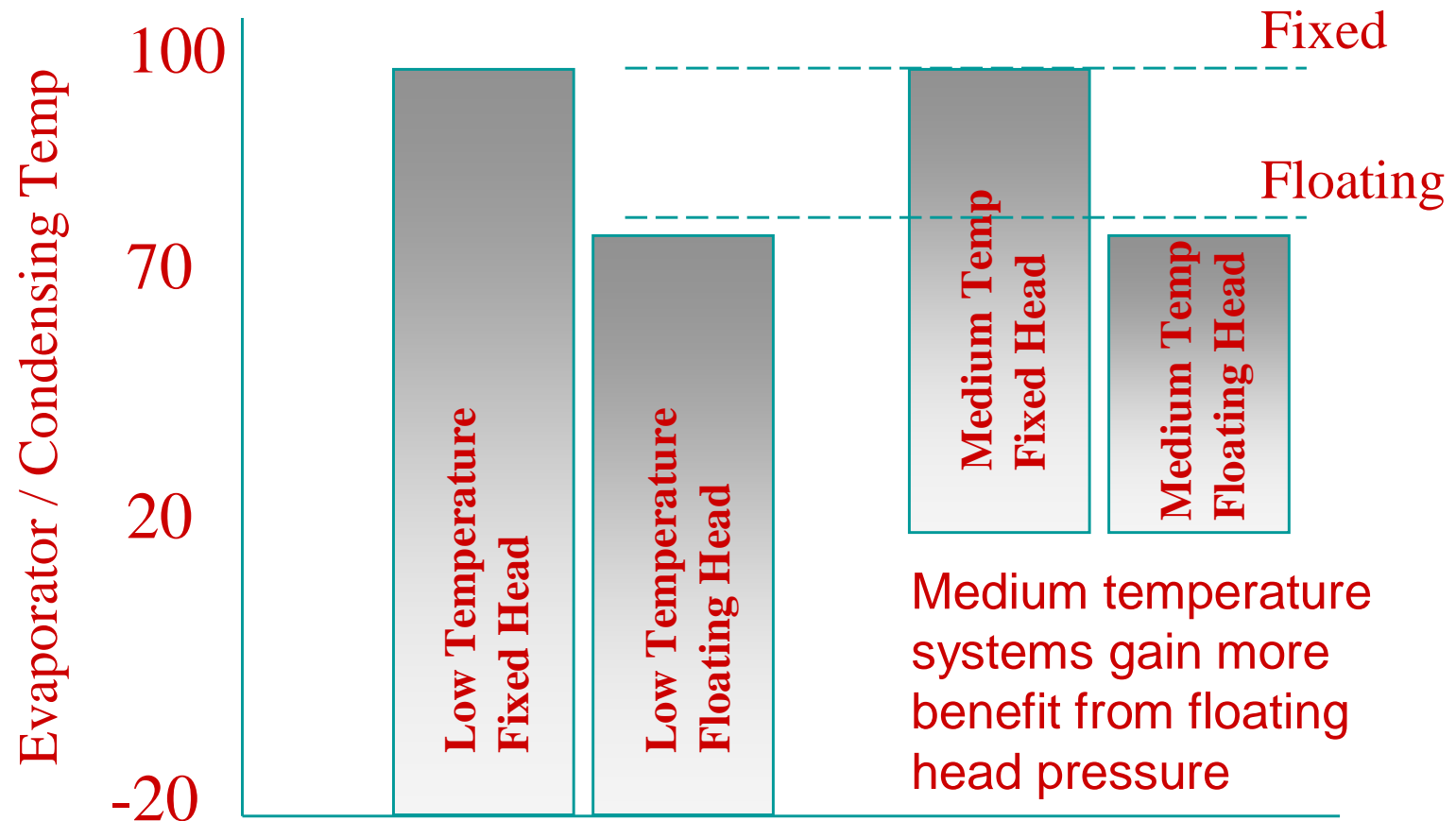




Industrial refrigeration Cost and Energy Efficiency Opportunities

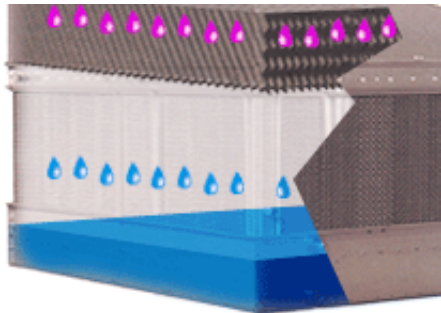
- Equipment selection
 - Component Selection to Maximize Tons/kW
 - Sizing: base design on actual loads vs transient values
- Minimize “lift”:
 - Floating suction pressure, floating head pressure
- Variable speed – various applications
- System design focused on:
 - Reliable Proven Equipment and Design Practices
 - Control for Off-design conditions
 - Ease of Service and Low Maintenance Costs
 - Prepiped Packaged Solutions

Refrigeration system “lift”

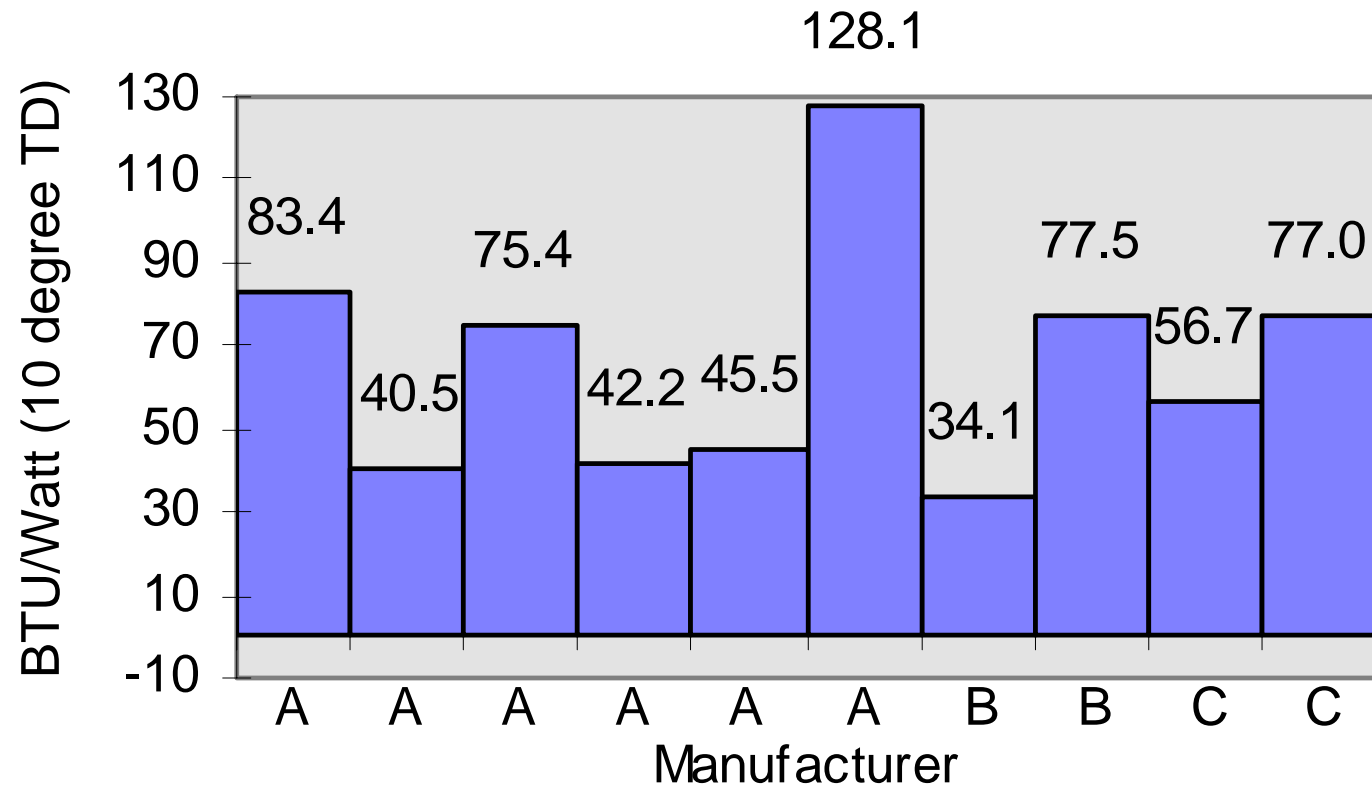


Condensers – design and selection practice

- **Nominal sizes stem from HVAC practice**
- **Air cooled condensers**
 - ◆ capacity varies with ambient Dry Bulb Temp
 - ◆ past practice driven by limiting maximum pressures
 - ◆ sizing TD of 10 F (LT) to 15 F (MT) unchanged for years
 - ◆ but, big range in motor size
- **Evap cooled condensers**
 - ◆ capacity varies with Wet Bulb Temp
 - ◆ past industry practice based on first cost (95 F SCT)
 - ◆ TD declining over time: 25° >> 16° >> 10°(?)
 - ◆ big range in fan power
- **Wide range of catalog efficiencies**



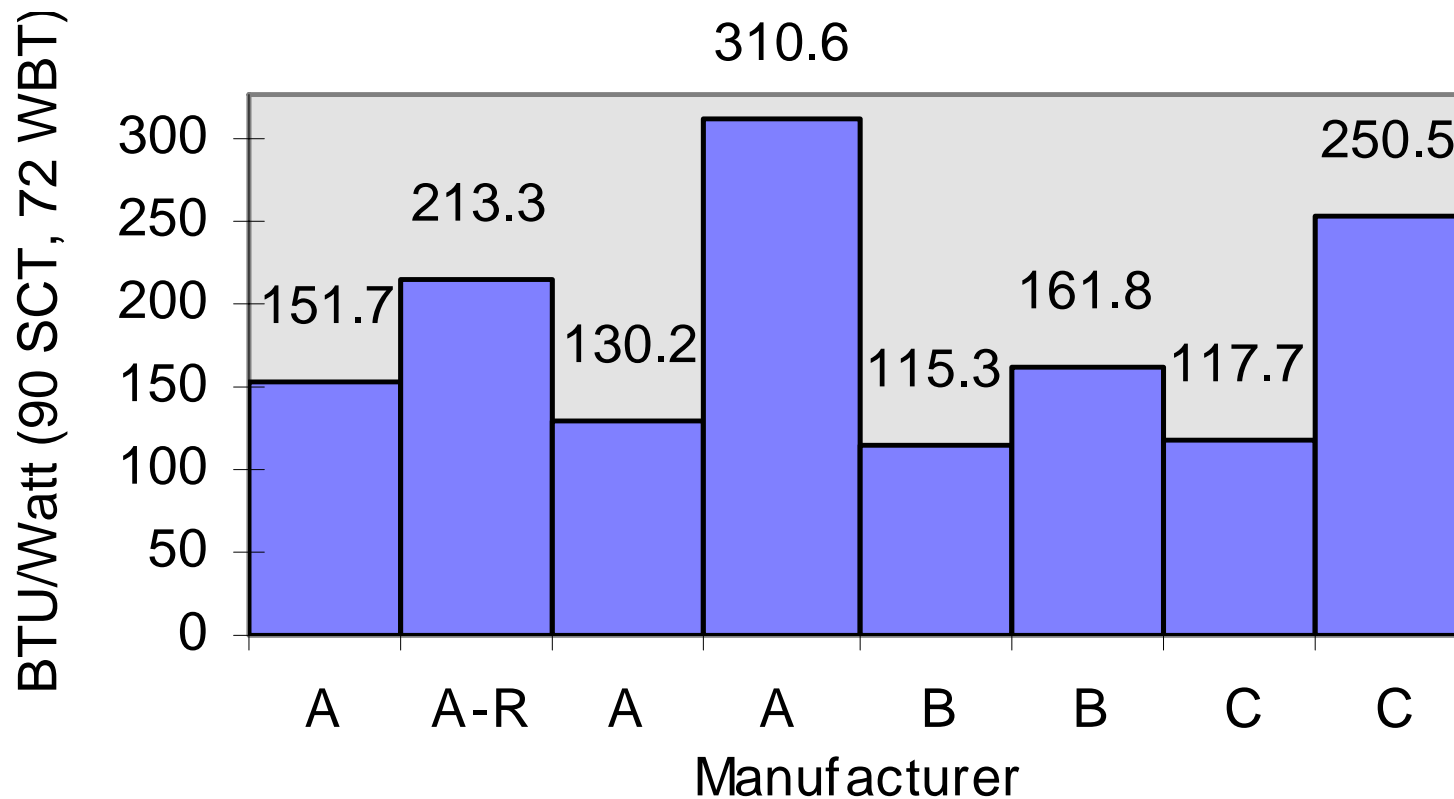
Variation in efficiency – air cooled



Examples of air cooled condenser specific efficiency



Variation in efficiency – evap cooled



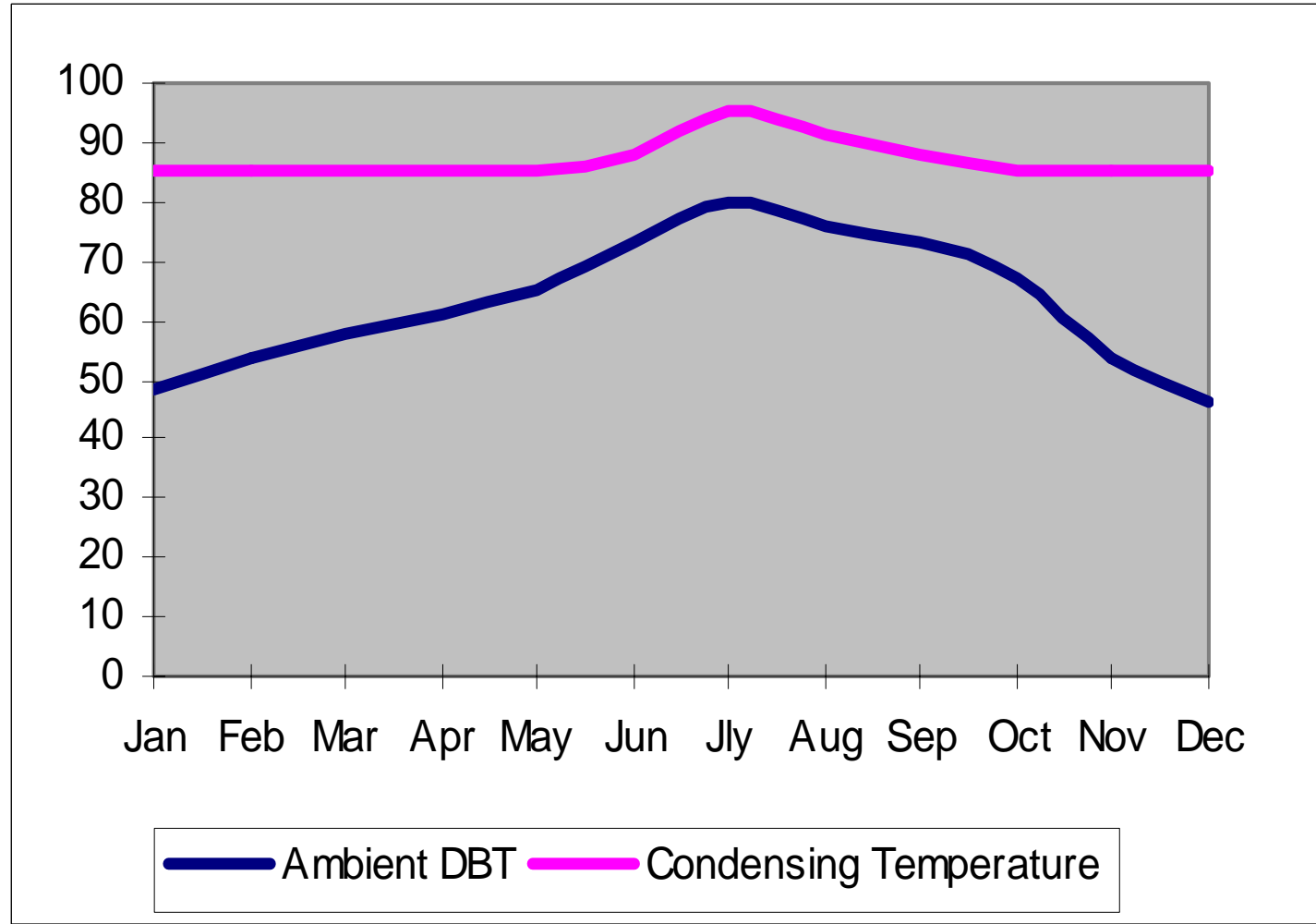
Examples of evap cooled condenser specific efficiency

Heat Rejection Control: Floating Head Pressure and Variable Speed Application

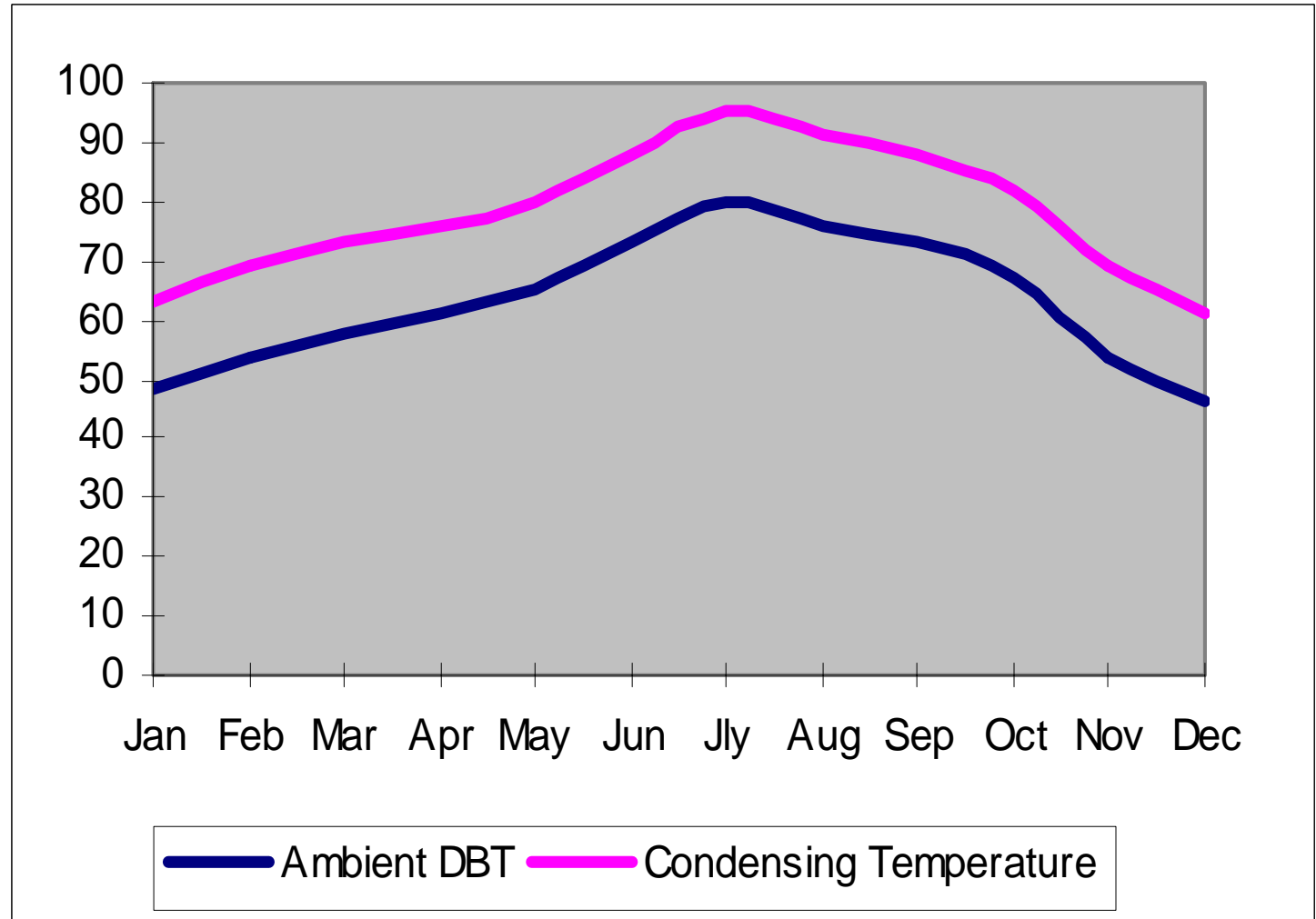




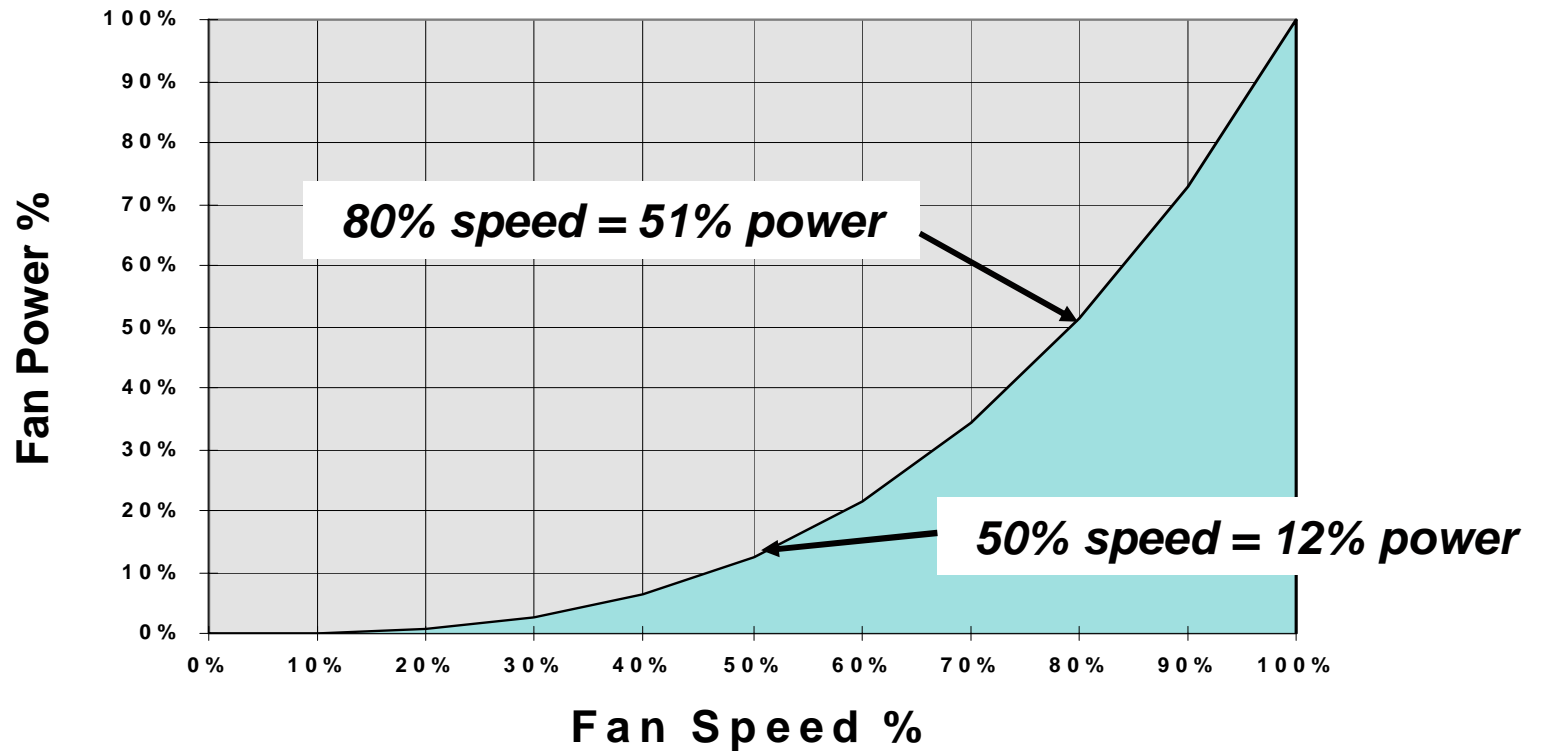
Fixed head pressure



Floating head pressure



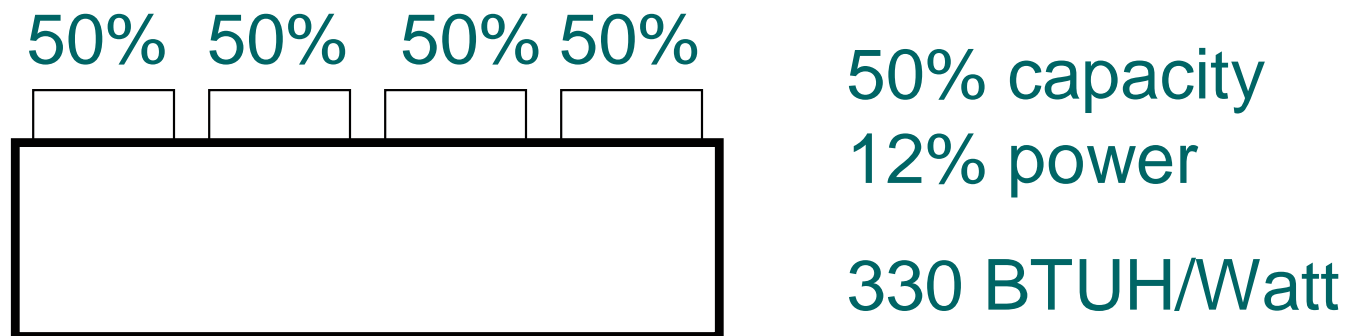
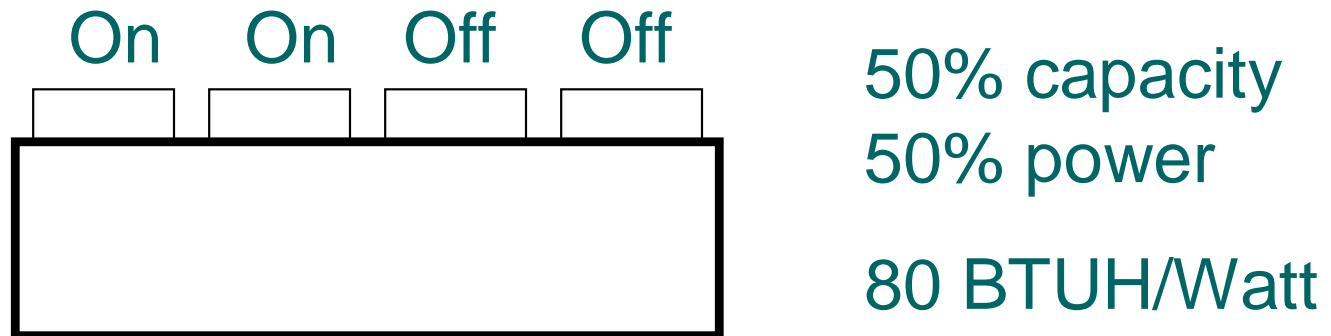
Variable speed fan control – third power relationship



Capacity varies directly with change in airflow

Fan power varies with cube of change in airflow

Part load condenser performance variable speed vs. fan cycling



Specific efficiency increased by 300% with variable speed

Floating head pressure common challenges



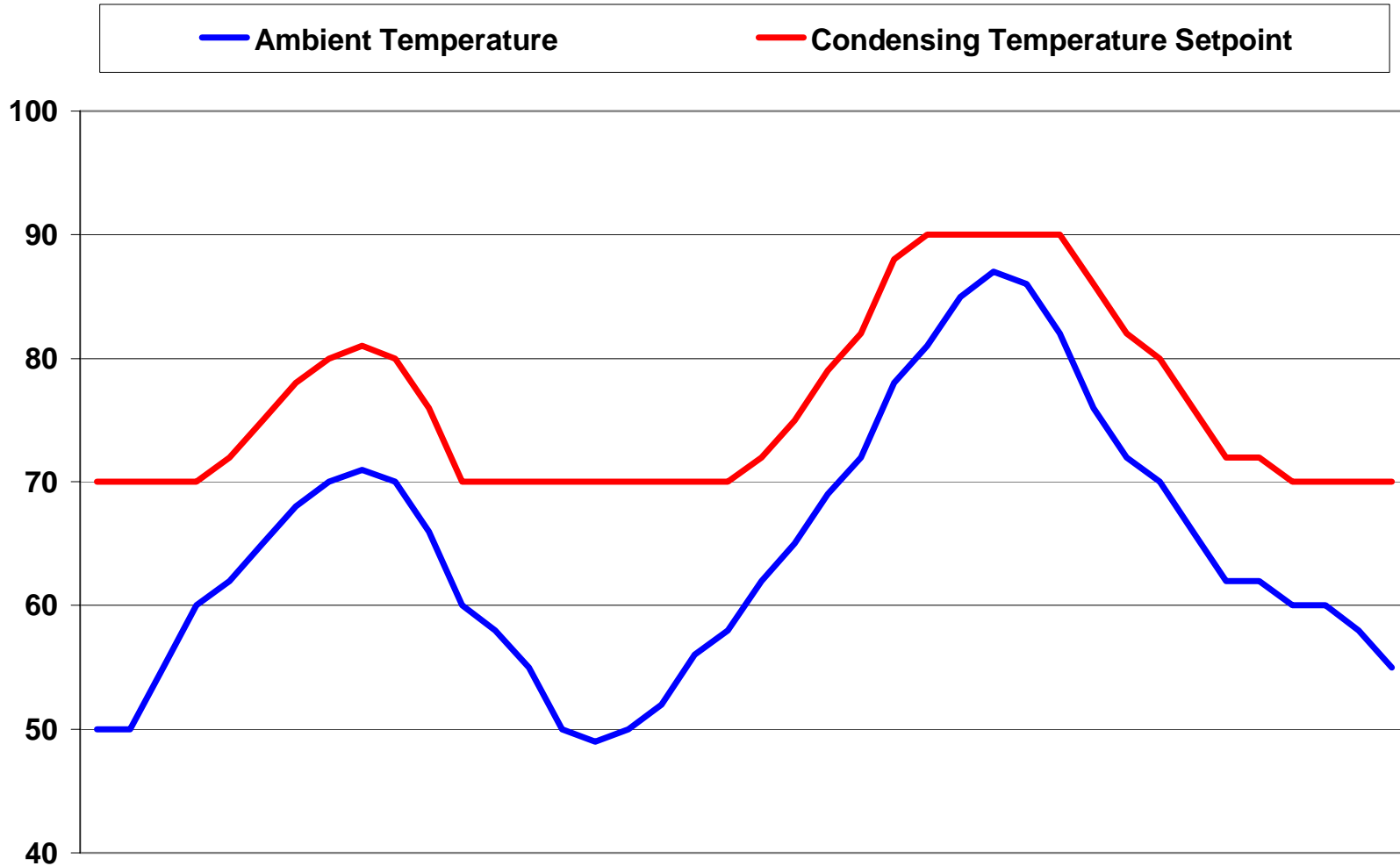
- **Large benefit even in moderate climates**
- **No brainer? Why not more widely adopted?**
- **Potential and/or perceived problems:**
 - ◆ may require more refrigerant charge
 - ◆ erratic system operation, liquid feed problems
 - ◆ system has too much capacity (at wrong time)
 - ◆ oil separator velocity
 - ◆ Defrost/ice harvest problems
- **Problem often NOT lower pressure itself, rather the effect of larger fluctuations**
- **Requires understanding overall system design and engineering considerations**

Integrated heat rejection control – floating head pressure successfully

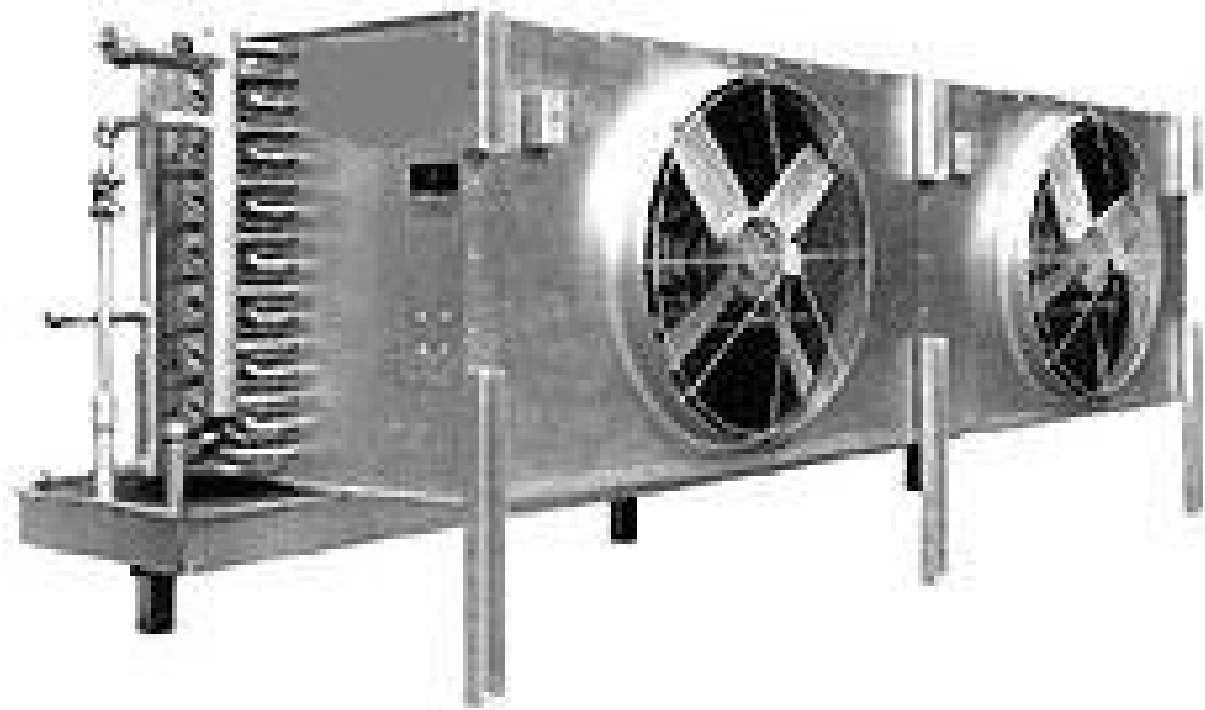


- **Condenser fan control, including:**
 - ◆ variable speed fan
 - ◆ variable setpoint control (change setpoint with weather)
- **Floating head pressure**
 - ◆ savings with lower head pressure
 - ◆ savings with steady head pressure
- **Standardized approach**
 - ◆ consistent hardware configuration
 - ◆ consistent strategy
 - ◆ consistent back-up operation
- **Must be serviceable and understandable**

Variable setpoint FHP



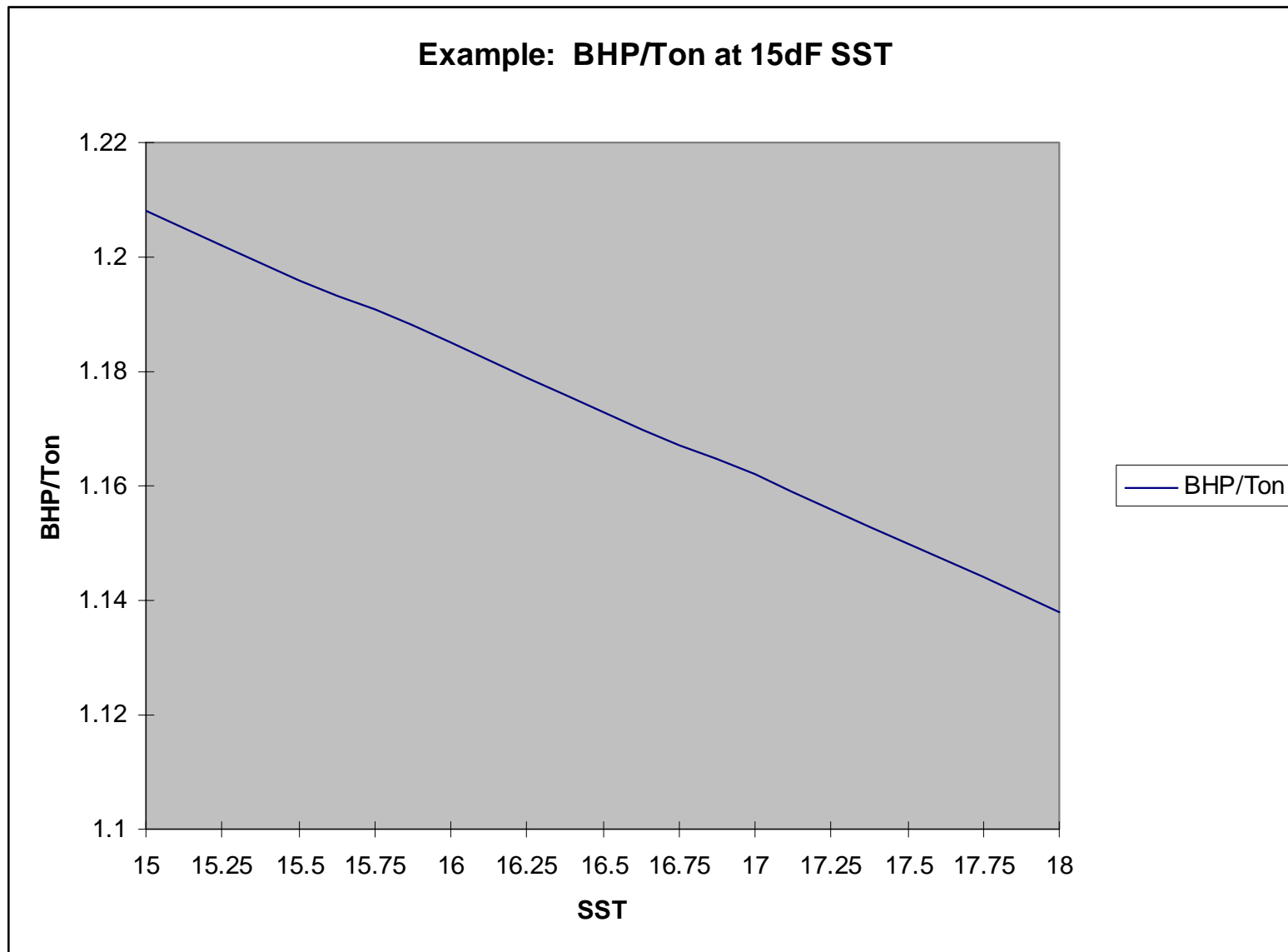
Air Unit Capacity Control



Floating Suction Pressure

- **Use your lowest temperature room as the control point.**
- **Use PID loop: As the temperature approaches the setpoint, incrementally raise the suction setpoint until stabilized. If the temp goes above setpoint, slowly lower the SSP.**
- **Saving in:**
 - ◆ Compressor loading due to increase efficiency
 - ◆ Reduces temperature variations
- **Make small changes, long intervals**
- **Excellent for storage warehousing**

3dF Maximum Float: Average Efficiency Gain of 2.9%





Variable volume variable temperature air unit control

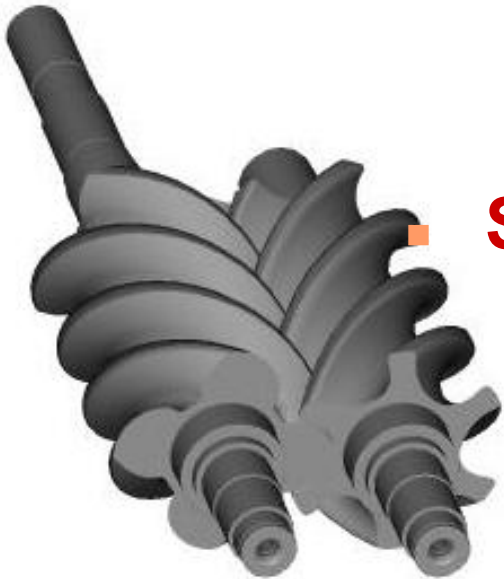
- **Vary fan speed in freezers and coolers as primary means of temperature control**
- **Saving in:**
 - ◆ Fan energy
 - ◆ Refrigeration cooling load (at high kW/Ton)
- **Third power rule applies to fan power**
- **Reduce speed to 60-70% then float suction**
- **Typical concerns and response:**
 - ◆ Motors burn up: *use right motor, don't run too slow*
 - ◆ Air falls on the floor: *education, don't run too slow*
 - ◆ Coils won't feed, won't defrost: *don't run too slow, try it*
 - ◆ *Not good for rooms with high product pulldown req.*

Variable Speed Screw Compressor Capacity Control



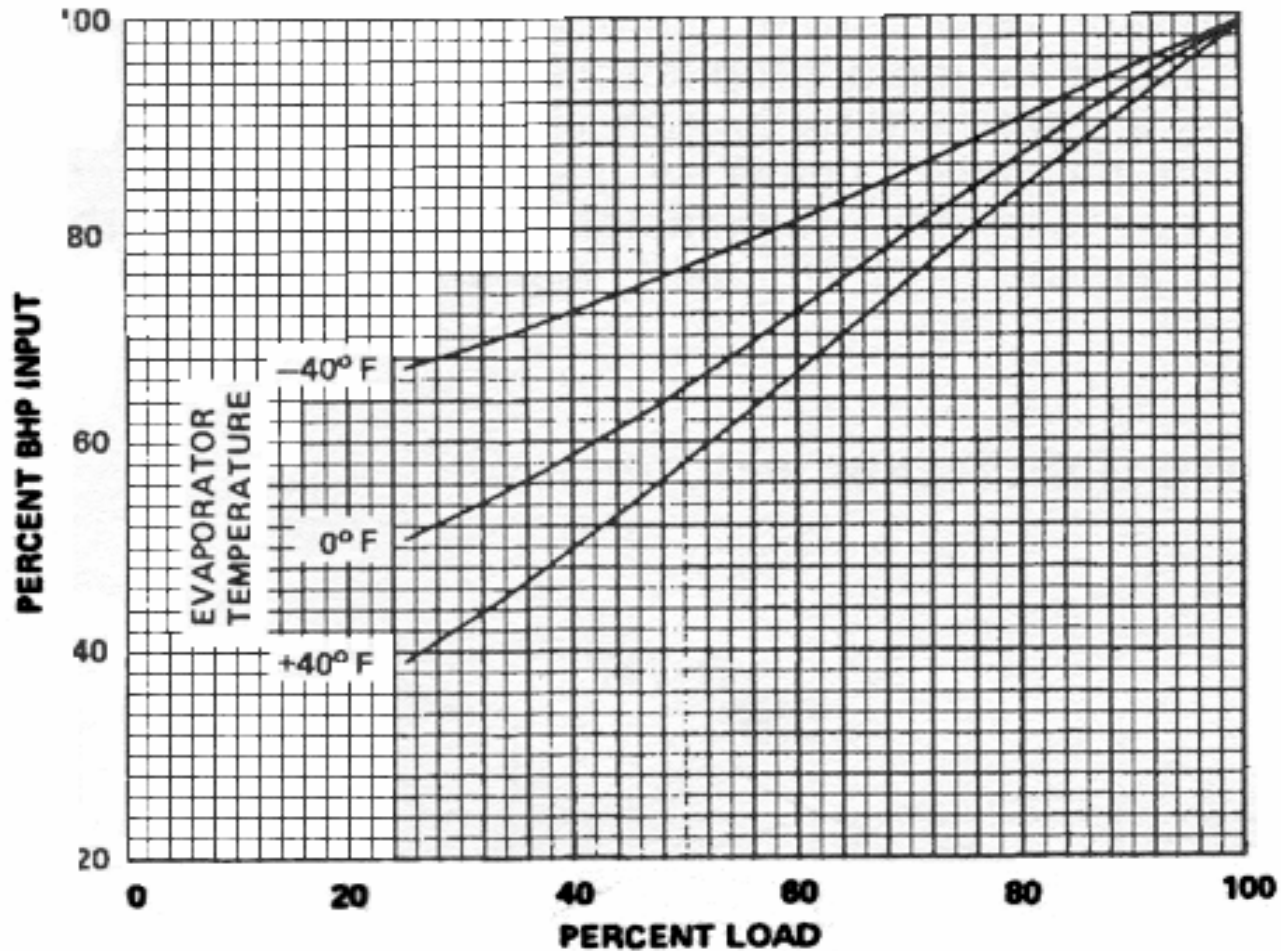
Variable speed screw compressor control

- Application does not! follow third power rule
- Minimum speed: approx. 50% (Frick = 20%)
 - ◆ Use slide valve below 50%
- Losses:
 - ◆ Increased leakage due to lower tip speed
 - ☞ Minimal losses, based on manufacturers software
 - ☞ Varies with application condition
 - ◆ VFD losses: fixed and variable components, ~4% total
- Savings:
 - ◆ Depends on time at reduced capacity
 - ◆ Requires adequate control sophistication (overall plant)
 - ◆ Manufacturers data varies A LOT



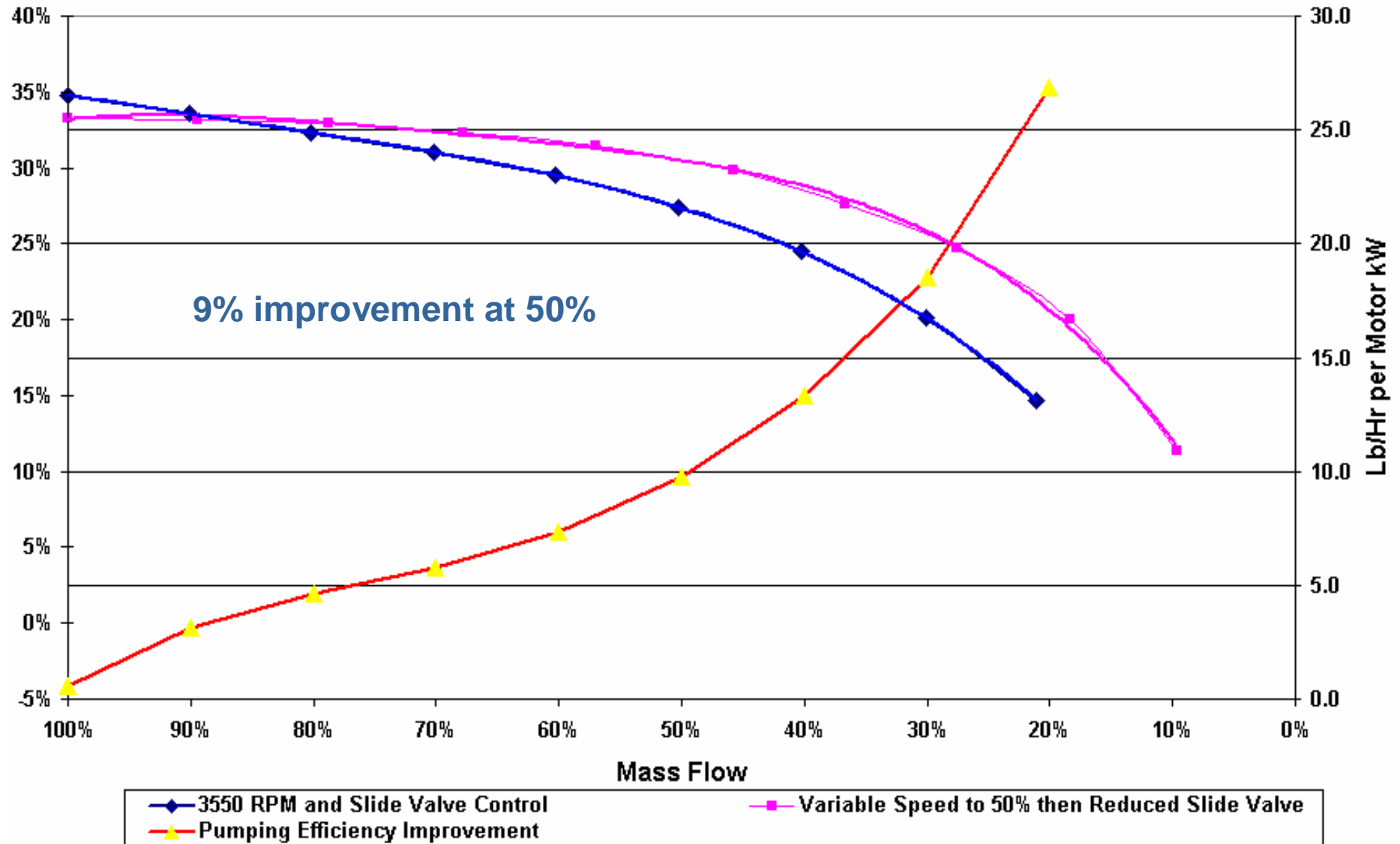
Capacity vs. BHP at part load

TYPICAL PART LOAD POWER INPUT WITH CONSTANT CONDENSING TEMPERATURE - HIGH STAGE



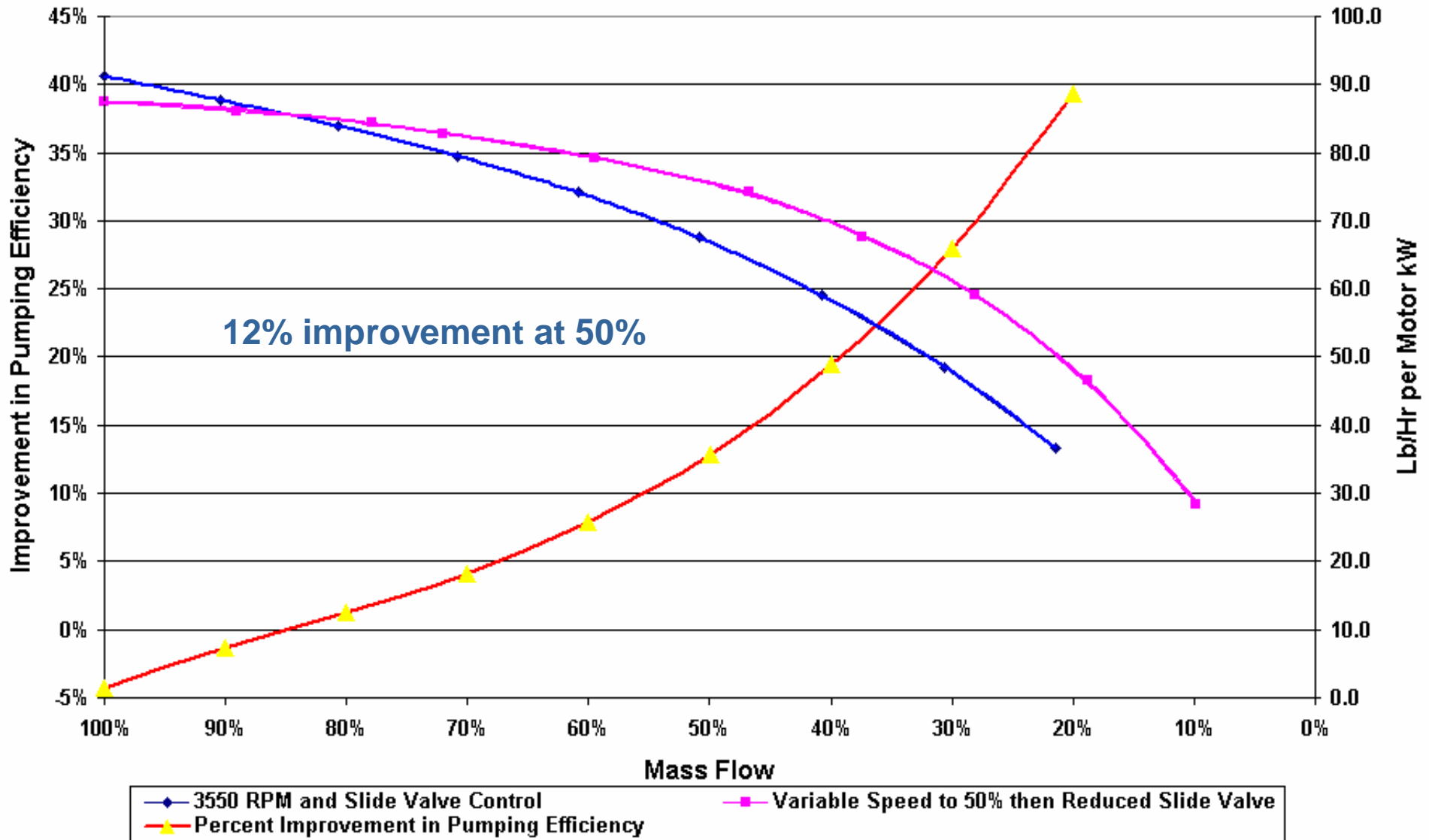
Medium temp VFD vs. slide valve

Medium Temp 200 HP R-717 at +15 SST and 90 SCT (with drive losses)



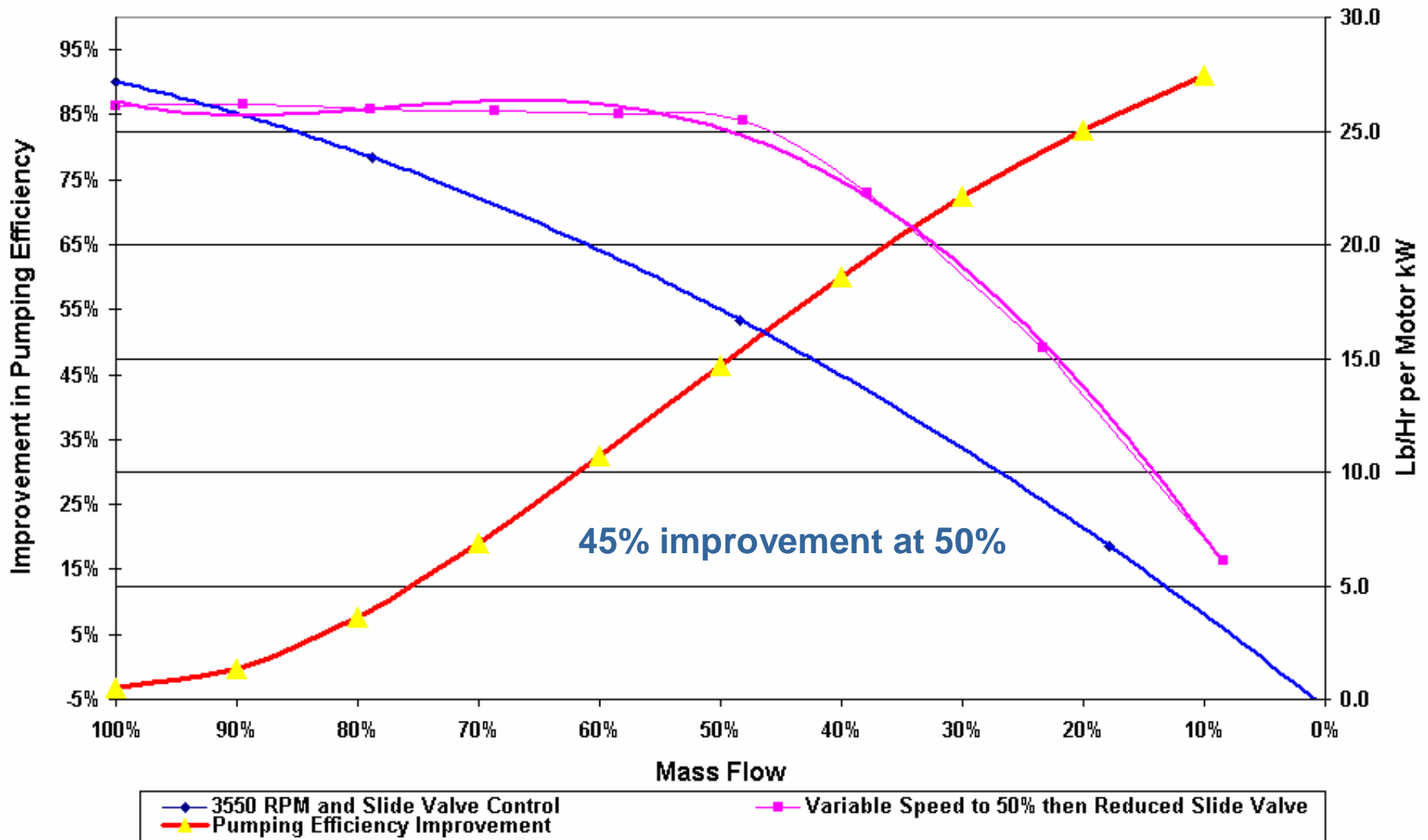
Low temp VFD vs. slide valve

Low Temp 125 HP Single Stage R-22 at -25 SST and 90 SDT (with drive losses)



LT booster VFD vs. slide valve

Booster 125 HP R717 at -35 SST and 20 SDT (with drive losses)



Compressor Control Strategies

- **Study the capacities and efficiencies of each compressor on a suction group.**
- **Bring on compressors that result in the smallest increment of capacity while maintaining 80% slide valve or more on the running compressors**
- **Unload in the same fashion**
- **In high demand areas, use offset suction pressure during the day, lower at night**

Demand Control in Industrial Refrigeration



Demand Management Issues

- Lower consumption (kWh) lowers demand (kW) passively.
- Active Demand Control will likely increase consumption.
- Dollar Savings Opportunities primarily during Peak Demand.
- Demand Costs vary by Utility and Rate Structure.
- Primary Benefit during Summer Months (June-October).
- Requires Active Schedule Change or Multiple Dated Control Screens.
- Demand Shifting vs Demand Control
- Some Experimentation Involved.
- User Specific. Must be Customized.
- Not Everyone can Benefit.

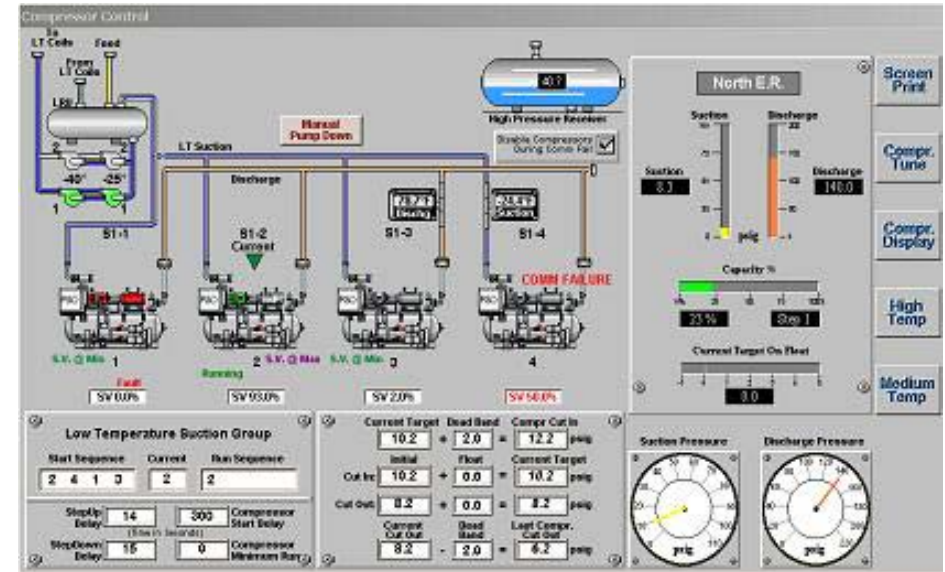


Demand Reduction Opportunities May Be Very Limited

- Three Shift Manufacturing Plants
- Process Production Plants
- Inadequate Existing Control Systems
- Seasonal Maximum Production Coincides with Peak Rates
- Insulated Envelope or Infiltration Loads Too Great
- Temperature or Humidity Requirements Too Stringent
- Limited Equipment Quantity (e.g. batteries for lifts)



Calculations:



- Simple spreadsheet to estimate savings, generally, within 10%.

Industrial refrigeration efficiency Looking forward/conclusions

- **Potential for improved part load performance**
 - ◆ improved load control strategies (central plants)
 - ◆ lower head pressures with optimized condenser control
 - ◆ variable speed compressor control (10-40% gains)
- **Performance information**
 - ◆ real time information – Internet based
 - ◆ tie to plant information, energy \$ per production unit
 - ◆ benchmark “ideal” performance for comparison
- **Other refrigeration opportunities**
 - ◆ More focus on evaporator coil/fan performance: need specific efficiency criteria & control methods
 - ◆ smaller systems can have high performance gains

Giving you what you want:

- Improved control and record keeping of refrigeration and temperature for OSHA, EPA, and USDA review.
- Reduction in plant maintenance due to lower speeds, lower compression ratios.
- Reduced Operating Expense = **MORE PROFIT!**

